Exact eddy-viscosity equation for turbulent wall flows

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What's new in this Version 0.135 of August 26, 2020/version 0.11 of June 18, 2020

- I wrote a 1st version of the introduction, the list of physical questions has been archived in the V0.11.
- I suppressed the former section 2 Flow cases and state of the art to follow Stefan's 'linear writing' concept; some elements have been reused in the new section 2.1.
- I changed the color codes for the Reynolds numbers in the figures 5,6,7,8,... now the curves for CF, $Re_{\tau} = 1995$ are in blue in the figures 3 and 6, etc.
- I modified the section 2.6 (former section 3.5) and its figures 9 by keeping only plots with y^+ as the abscissa; moreover I used smaller values of Re_{τ} accessible with DNS.
- I modified the section 2.8 (former section 3.7) and its figures 10 by including also asymptotics for $D_{\nu i}$ in the outer region; moreover I use lower Re_{τ} (1955 and 5186) in figures 10 since the convergence to the asymptotics is fast.
- I created the section 2.9 and correspondingly a new line in the figures 10.
- I created the section 2.10.
- I reworked entirely the section 3 (former section 4). The new Q7 and 8 in the sections 3.4 and 3.5 have been answered by Stefan!
- I created the appendices A and B.

Abstract (to be reworked)

A recent theory has been developed (Heinz 2018, 2019) for three canonical turbulent wall flows: channel flow, pipe flow and zero-pressure gradient boundary layer, that offers exact analytical formulas for the RANS eddy-viscosity, as a product of a function of y^+ (the wall-normal distance scaled in inner units) with functions of y/δ (the same distance scaled in outer units). By calculating the eddy-viscosity turbulent diffusion term for these flows where the turbulence is stationary, one identifies a high-Reynolds number RANS eddy-viscosity equation with one production and two dissipation terms. One dissipation term is universal, peaks in the near-wall region, and scales mainly with y^+ . The second one, smaller in magnitude, is flow-dependent, peaks in the wake region, and scales mainly with y/δ . The production term is flow-dependent, peaks in between, and scales also mainly with y/δ . The universal dissipation term implies a damping function and a length scale analogous to the von Karman length scale used in the Scale-Adaptative Simulation models. This length scale also appears in the production term. This confirms on firm theoretical bases the relevance of von Karman length scales. This is an occasion to analyze these length scales in more details. An asymptotic analysis of all terms in the eddy-viscosity budget in the limit of infinite Reynolds numbers is also proposed. This allows a review and tests of existing RANS models that imply an eddy-viscosity equation. Finally, we propose a new version of the eddy-viscosity equation of the Scale-Adaptative Simulation models.

1 Introduction

Reynolds-Averaged Navier-Stokes (RANS) models are still widely used in Engineering Computational Fluid Dynamics, because they allow studies in complex setups for a lower computational cost than more sophisticated methods like Large Eddy Simulations or Hybrid methods (Wilcox 2006; Hanjalić & Launder 2011). Among RANS models, twoequations models like the $k-\omega$ (Kolmogorov 1942; Wilcox 1988) and $k-\epsilon$ (Launder & Spalding 1974) models are popular. Their equation for the turbulent kinetic energy k is known to be rather accurate. Indeed, its high-Reynolds number form can be validated with direct numerical simulations (DNS - we use those of Lee & Moser 2015) in channel flows, out of the viscous and buffer layers, as shown in the appendix A. On the contrary, the equations for the turbulent dissipation ϵ or the turbulent frequency ω of these models are much less accurate, as shown, with the same DNS data, in the appendix B. These problems are related to the fact that these equations have been constructed empirically on phenomenological and dimensional bases. Alternate RANS models where one turbulent field is the eddy-viscosity ν_t itself, which evolves according to its own equation, look appealing since ν_t is doubtless a quite relevant variable. Whereas the first models of this kind also constructed the ν_t - equation on phenomenological and dimensional bases (Nee & Kovasznay 1969; Baldwin & Barth 1990; Spalart & Allmaras 1994), there have been recently attempts to use more systematic approaches (Yoshizawa et al. 2012; Hamba 2013). The Scale-Adaptive Simulation (SAS) models of Menter et al. (2006); Menter & Egorov (2010) are also $k - \nu_t$ models, which are more phenomenological than the ones of Yoshizawa et al. (2012); Hamba (2013), but may work as Hybrid models. They imply through the so-called von Karman length scale ℓ_{vK} the second gradient of the mean velocity, which renders them 'sensitive'. In highly nonhomogeneous flows the second gradient of the mean velocity, which appears in the denominator of ℓ_{vK} , increases, hence ℓ_{vK} decreases, hence ν_t decreases: this allows resolved motions in unsteady simulations. Relevant applications of variants of this model to quite complex flows are for instance presented in Menter et al. (2006); Egorov et al. (2010); Abdol-Hamid (2015). Jakirlic & Maduta (2015) also proposed an interesting extension of the SAS approach under the form of a Reynolds stress model.

The facts that many variants of the SAS models exist, and that their equations are built on phenomenological grounds, raise however theoretical questions. Accordingly, it is noticeable that the more systematic approaches of Yoshizawa et al. (2012); Hamba (2013) are quite different: whereas Yoshizawa et al. (2012) started from a conventional Reynolds-stress model, Hamba (2013) started from the nonlocal analysis presented in Hamba (2005). Obviously, there is no perfect way to analytically derive the ν_t - equation.

The aim of this work is to offer a third way, at least for a relevant class of flows: established channel and pipe flows, together with zero-pressure gradient boundary layers over a flat plate. For these three canonical flows, denoted hereafter 'turbulent wall flows', Heinz (2018, 2019) proposed analytic models of the mean flow U, main Reynolds stress $-\langle u_x u_y \rangle$ and eddy viscosity ν_t , built after a thorough analysis of recent DNS, including those of Lee & Moser (2015); Chin et al. (2014); Sillero et al. (2013), and experimental data, for instance those of Vallikivi et al. (2015). One interest of these models is that they are valid for friction-based Reynolds numbers $Re_\tau \gtrsim 500$: the limit $Re_\tau \to \infty$ is included. Our approach is then to analytically calculate the turbulent diffusion of ν_t and to identify the opposite of this as the sum of a positive production term plus negative dissipation terms, see equation (12). The fact that

these terms are analytical offers a much better intellectual understanding of the ν_t - equation, and also a practical understanding of the scaling properties of all terms in this equation. This also permits an accurate description of the limit $Re_{\tau} \to \infty$. Another goal is the creation of a basis for the numerical and physical evaluation of existing models.

In section 2, we offer a presentation and physical analysis of our model, which considers only high-Reynolds number turbulent wall flows. The physical questions that we want to answer concern the scalings of the production and dissipation terms in the ν_t - equation, and the effects of the flow cases and Reynolds number. Asymptotic formulas valid in the limit $Re_{\tau} \to \infty$, for all terms in the ν_t - equation, are in particular given in the sections 2.6 to 2.9. An overview of all our physical results closes in the section 2.10.

An evaluation of existing models is performed in the section 3.

...

2 Analysis: exact eddy-viscosity formula and transport equation

2.1 Turbulent wall flows - Exact eddy viscosity formula

Following Heinz (2018, 2019), we consider turbulent wall flows of incompressible fluids of mass density ρ and kinematic viscosity ν . Locally a cartesian system of coordinates Oxyz is used, such that x points in the streamwise direction, and y measures the distance to the closest wall. To lowest order, the mean flow

$$\mathbf{U} = U(y) \mathbf{e}_x \tag{1}$$

with \mathbf{e}_x the unit vector in the x-direction. A relevant quantity is the mean strain rate

$$S = \partial U/\partial y . (2)$$

The macroscopic length scale δ is the half-channel height, pipe radius, or 99% boundary layer thickness with respect to channel flow, pipe flow, and boundary layer, respectively. Denoting $u_x \mathbf{e}_x + u_y \mathbf{e}_y + u_z \mathbf{e}_z$ the fluctuating velocity, the RANS eddy viscosity

$$\nu_t = -\left\langle u_x u_y \right\rangle / S \tag{3}$$

where the angular brackets denote the Reynolds average. The mean wall shear stress τ_w is used to define the friction velocity $u_{\tau} = \sqrt{\tau_w/\rho}$. From this are defined wall or inner units: $y^+ = u_{\tau}y/\nu$, $U^+ = U/u_{\tau}$ and

$$S^{+} = \partial U^{+}/\partial y^{+} . (4)$$

Finally, the friction-velocity Reynolds number $Re_{\tau} = \delta^{+} = u_{\tau}\delta/\nu$. In the equation (11) of Heinz (2019), an analytic expression is proposed for the reduced eddy viscosity, which is valid at high Reynolds number, $Re_{\tau} \gtrsim 500$,

$$\nu^{+} = \nu_{t}/\nu = (1/S_{12}^{+} - 1) W .$$
 (5)

There $S_{12}^+ = S_1^+ + S_2^+$ is a very good approximation of the dimensionless mean strain rate S^+ (4) in the inner region of the flows, i.e., disregarding wake effects, see the equation (7) of Heinz (2018) and the corresponding discussion. Precisely, the universal function

$$S_{12}^{+} = S_{12}^{+}(y^{+}) = 1 - \left[\frac{(y^{+}/a)^{b/c}}{1 + (y^{+}/a)^{b/c}} \right]^{c} + \frac{1}{\kappa y^{+}} \frac{1 + h_{2}/(1 + y^{+}/h_{1})}{1 + y_{k}/(y^{+}H)} , \tag{6}$$

with

$$a = 9, b = 3.04, c = 1.4, H = H(y^{+}) = (1 + h_1/y^{+})^{-h_2}, h_1 = 12.36, h_2 = 6.47, y_k = 75.8,$$
 (7)

and the von Karman constant

$$\kappa = 0.40. \tag{8}$$

The function S_{12}^+ , plotted on the figure 1a, approaches naturally 1 as $y^+ \to 0$ in the viscous sublayer. On the contrary, as $y^+ \to \infty$, $S_{12}^+ \sim 1/(\kappa y^+)$, in agreement with the log law. Therefore the function $1/S_{12}^+ - 1$, plotted on the figure 1b, which appears in the eddy viscosity (5), vanishes in the limit $y^+ \to 0$, and then increases smoothly to approach the function κy^+ as $y^+ \to \infty$.

The second ingredient of the theory is the function W, which is flow-dependent and in outer scaling, because it describes wake effects. With the notations of Heinz (2018, 2019), $W = 1/G_{CP}$ for channel and pipe flows, M_{BL}/G_{BL} for boundary layers, where G_{CP} and G_{BL} characterize the wake contribution S_3^+ to the dimensionless mean strain

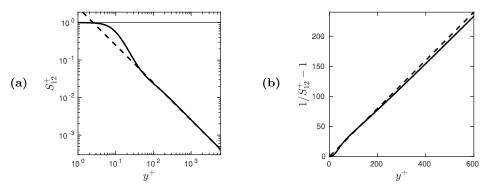


Fig. 1: (a) Continuous line: S_{12}^+ , dashed line: $1/(\kappa y^+)$. (b) Continuous line: $1/S_{12}^+ - 1$, dashed line: κy^+ .

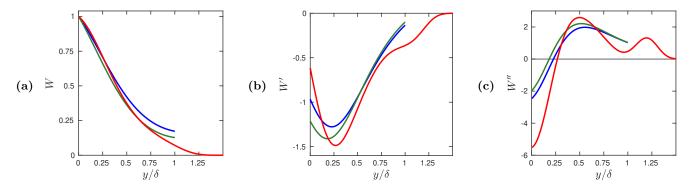


Fig. 2: (a) W (b) W' (c) W'' for channel (blue), pipe (green), boundary layer (red).

rate S^+ (see the equations 7 and A.22 of Heinz 2018), M_{BL} characterizes the total stress in boundary layers (see the equation 4 of Heinz 2019). For channel and pipe flows

$$W = W_X(y/\delta) \quad \text{with} \quad W_X(y) = \frac{K_X y + (1-y)^2 (0.6y^2 + 1.1y + 1)}{1 + y + y^2 (1.6 + 1.8y)}, \tag{9}$$

X = C, $K_C = 0.933$ for channel, X = P, $K_P = 0.687$ for pipe; for boundary layers

$$W = W_{BL}(y/\delta) \quad \text{with} \quad W_{BL}(y) = \frac{1 + 0.285 \ y \ e^{y(0.9 + y + 1.09y^2)}}{1 + (0.9 + 2y + 3.27y^2)y} \ e^{-y^6 - 1.57y^2} \ . \tag{10}$$

The wake function W is plotted for these three flows on the figure 2a. In the near-wall region, when $y/\delta \to 0$, $W \to 1$, hence the eddy viscosity (5), $\nu^+ = (1/S_{12}^+(y^+) - 1) \, W(y^+/\delta^+) \sim (1/S_{12}^+(y^+) - 1)$ where $\delta^+ = Re_\tau$. Therefore the log-layer eddy viscosity κy^+ is approximately recovered if $1 \ll y^+ \ll \delta^+$; for a more precise study, see the section 4.1 of Heinz (2019). When y becomes of the order of δ , wake effects come in, that saturate the growth of the eddy viscosity (5), since W decreases. Whereas the maximum value of y is δ in channel and pipe flows (if $y \in [\delta, 2\delta]$ the mean fields can be obtained by suitable symmetries from the mean fields for $y \in [0, \delta]$), y may be much larger in boundary layers. Naturally, $W_{BL} \to 0$ as $y \to \infty$; precisely $W_{BL} < 10^{-3}$ as soon as $y > 1.36\delta$.

This model has been validated by a study of DNS and experimental data. For instance, the figures S.6abc of the supplementary material to Heinz (2019) show the eddy viscosity of various DNS, one for each canonical flow, compared with two variants of the eddy-viscosity model (5). In particular, the magenta curves show $\nu^+ = \kappa y^+ W$ with our notations, i.e. $(1/S_{12}^+ - 1)$ in (5) has been replaced by κy^+ . The agreement with the DNS is good, except in the outer region, where in (3) both the numerator $\langle u_x u_y \rangle$ and the denominator S tend to zero, hence the DNS noise is amplified.

Since the derivatives W' and W'' will be needed hereafer, they are plotted on the figures 2bc. Whereas the functions W for the three flow cases are quite similar (figure 2a), their first and second derivatives show larger differences (figures 2bc). Naturally, W'_{BL} and $W''_{BL} \to 0$ as $y \to \infty$.

2.2 Exact eddy-viscosity equation for turbulent wall flows

Since the focus of our study is on high-Reynolds numbers wall-bounded flows, we assume that the eddy-viscosity equation reads

$$\sigma \frac{\partial \nu_t}{\partial t} = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \nu_t}{\partial y} \right) + P_{\nu} - D_{\nu} \tag{11}$$

with $P_{\nu} > 0$ the production, $D_{\nu} > 0$ the dissipation term. The dimensionless coefficient σ , of order 1, which is a 'turbulent Prandtl number', plays no role in the turbulent wall flows, where the mean fields are steady, but is kept in (11) for the sake of comparison with existing turbulence models. In turbulent wall flows, according to (11), the opposite of the turbulent diffusion term

$$-T_{\nu} = -\frac{\partial}{\partial y} \left(\nu_{t} \frac{\partial \nu_{t}}{\partial y} \right) = P_{\nu} - D_{\nu} . \tag{12}$$

A formal computation of T_{ν} starting from (5) leads to $D_{\nu} = D_{\nu i} + D_{\nu o}$ and

$$D_{\nu i} = \kappa^2 \frac{\nu_t^2}{L_{vK}^2} \frac{1}{f^2} \,, \tag{13a}$$

$$P_{\nu} = \kappa \frac{\nu_t^2}{L_{vK} \delta} \frac{1}{1 - S_{12}^+} \left(-\frac{4W'}{W} \right) = \kappa \frac{\nu \nu_t}{L_{vK} \delta} \frac{1}{S_{12}^+} \left(-4W' \right), \tag{13b}$$

$$D_{\nu o} = \frac{\nu_t^2}{\delta^2} \frac{W'^2 + WW''}{W^2} = \frac{\nu^2}{\delta^2} (1/S_{12}^+ - 1)^2 (W'^2 + WW'') . \tag{13c}$$

The indices i and o refer to 'inner' and 'outer' terms, respectively, and the notation $D_{\nu o}$ is slightly improper since this term is slightly negative in the near-wall region. However, $D_{\nu o}$ is much smaller in this region than in the outer region where it peaks, as it will be shown in the figure 6b for channel flow, 7b for pipe flow, 8b for boundary layers. Moreover $D_{\nu} = D_{\nu i} + D_{\nu o} > 0$ everywhere, as it will be shown in the figures 6cd for channel flow, 7cd for pipe flow, 8cd for boundary layers, hence the notation D_{ν} is fully justified.

In addition to the functions S_{12}^+ and W defined in the section 2.1, there appears in the equations (13) other functions that are built on these. The first one is the asymptotic von Karman length scale

$$L_{vK} = \kappa \left| \frac{S_{12}}{\partial S_{12}/\partial y} \right| \quad \text{or} \quad L_{vK}^+ = \kappa \left| \frac{S_{12}^+}{\partial S_{12}^+/\partial y^+} \right|.$$
 (14)

It is defined as the von Karman length scale used by the SAS models

$$\ell_{vK} = \kappa \left| \frac{S}{\partial S/\partial y} \right|, \tag{15}$$

but replacing S by S_{12} , i.e., disregarding 'wake effects'. The fact that the length scale L_{vK} appears in (13a) and (13b) confirms on very firm bases the relevance of this length scale, which was not so clear in the derivation of Menter et~al.~(2006); Menter & Egorov (2010). Only the inner-units $L_{vK}^+(y^+)$ is universal, whereas the physical $L_{vK}(y/\delta)$ has to be calculated as $\delta(L_{vK}^+/\delta^+)$, i.e. L_{vK}/δ depends on $\delta^+ = Re_\tau$. Since, as $y^+ \to \infty$, in agreement with the log law, $S_{12}^+ \sim 1/(\kappa y^+)$, $L_{vK}^+ \sim \kappa y^+$, as confirmed by the figure 3a. The functions $\ell_{vK}^+(y^+)$ (figure 3a) or $\ell_{vK}(y/\delta)$ (figure 3b), that depend on the flow case and Reynolds-number, have been computed using the accurate expressions of S^+ of the equation (7) of Heinz (2018), that take into account wake effects. In channel or pipe flow, U presents a maximum at the centerplane or pipe axis $y = \delta$, hence S and ℓ_{vK} vanish there. On the contrary, in boundary layer flow, S and ℓ_{vK} vanish only in the limit $y \to \infty$. The figure 3c suggests that, because the dimensional factor in D_{vi} (13a), P_v (first expression in 13b) and D_{vo} (first expression in 13c) are respectively ν_t^2/L_{vK}^2 , $\nu_t^2/(L_{vK}\delta)$ and ν_t^2/δ^2 , in the ratii δ^2/L_{vK}^2 , δ/L_{vK} , 1, those will peak in the inner, intermediate and outer regions; this will be confirmed in the figures 6 for channel flow, 7 for pipe flow, 8 for boundary layers.

Another ingredient in $D_{\nu i}$ (13a), is the universal damping function

$$f = f(y^{+}) = (1 - S_{12}^{+}) \left(\frac{(S_{12}^{+} - 1) S_{12}^{+} d^{2} S_{12}^{+} / dy^{+2}}{(dS_{12}^{+} / dy^{+})^{2}} + 3 - 2S_{12}^{+} \right)^{-1/2}.$$
 (16)

It is plotted on the figures 4ab. It does tend to zero as $y^+ \to 0$ and 1 as $y^+ \to \infty$.

Finally, in P_{ν} (13b) and $D_{\nu o}$ (13c) the rightmost functions depend only on W and its derivatives. In P_{ν} there appears -4W' which is positive according to the figure 2b, hence $P_{\nu} > 0$ as required. In $D_{\nu o}$ there appears $W'^2 + WW''$ which is plotted on the figure 4c. As already suggested after the equations (13), the function $W'^2 + WW'' > 0$ except in a more or less narrow near-wall region, depending on the flow case.

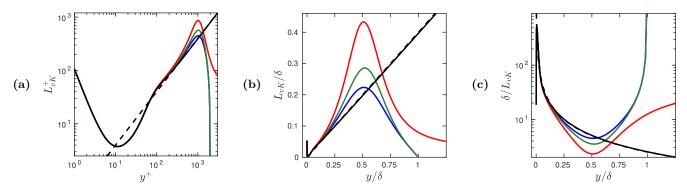


Fig. 3: (a,b) The asymptotic von Karman length scale L_{vK} (14) (black continuous); its log law approximation κy (black dashed); the von Karman length scale ℓ_{vK} (15) for channel (blue), pipe (green), boundary layer (red). The ℓ_{vK}^+ curves of the figure (a) and all curves of the figure (b) have been computed at $Re_{\tau} = 1995$. The figure (c) shows the same curves as figure (b) but with the inverse ordinates and linear-log scales. All curves in (a,b,c) start at $y^+ = 1$.

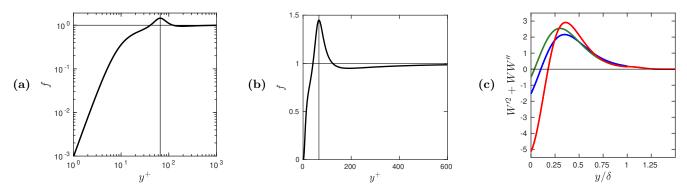


Fig. 4: (a,b) The damping function f. The vertical lines are at $y^+ = 66$. (c) The function $W'^2 + WW''$ for channel (blue), pipe (green), boundary layer (red).

2.3 Application to channel flows

In typical channel flow cases, a comparison of the opposite of the dimensionless turbulent diffusion term

$$-T_{\nu}^{+} = -\frac{\partial}{\partial y^{+}} \left(\nu^{+} \frac{\partial \nu^{+}}{\partial y^{+}} \right) = -\frac{T_{\nu}}{u_{\tau}^{2}} \tag{17}$$

computed with finite differences from two DNS of Lee & Moser (2015) and its model (12,13),

$$-T_{\nu}^{+} = P_{\nu}^{+} - D_{\nu}^{+} = -D_{\nu i}^{+} + P_{\nu}^{+} - D_{\nu o}^{+}$$

$$\tag{18}$$

is shown on the figures 5. Except in the outer region, where the DNS noise is amplified, there is a good agreement between the model and the DNS, especially, for the highest Reynolds number case.

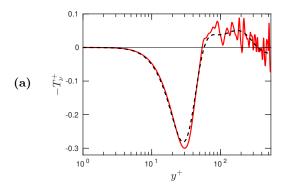
The separation of $-T_{\nu}^{+}$ into the three terms of the model, $-D_{\nu i}^{+}$, P_{ν}^{+} and $-D_{\nu o}^{+}$, is illustrated on the figures 6. The comparison of the figures 6a, c and g shows that the dissipation term $D_{\nu i}^{+}$ dominates in the near-wall region. In this region, and in inner scalings, $D_{\nu i}^{+}(y^{+})$, $D_{\nu}^{+}(y^{+})$ and $T_{\nu}^{+}(y^{+})$ approach as $Re_{\tau} \to \infty$ a limit profile, with a maximum around $y^{+} = 31$ and a minimum around $y^{+} = 72$. A plateau around $y^{+} \simeq 300$ and

$$D_{\nu i}^+ \simeq D_{\nu}^+ \simeq T_{\nu}^+ \simeq \kappa^2$$

builds up as $Re_{\tau} \to \infty$, in agreement with the formula for the log-layer reduced eddy viscosity, $\nu^+ = \kappa y^+$. For larger values of y/δ , after this plateau, the figures 6bdfh show that all terms, considered in inner-outer scalings, $D_{\nu i}^+(y/\delta)$, $D_{\nu o}^+(y/\delta)$, $D_{\nu o}^+(y/\delta)$, $D_{\nu o}^+(y/\delta)$, and $T_{\nu}^+(y/\delta)$, approach limit profiles as $Re_{\tau} \to \infty$.

2.4 Application to pipe flows

In the eddy-viscosity model (5), the only difference between channel and pipe flows is described by the change of the coefficient K_X in the function W_X (9) that contains the wake effects. This change from $K_C = 0.933$ to $K_P = 0.687$ is moderate, therefore the turbulent diffusion term and its contributions are close to the ones of channel flow, as shows the comparison between the figures 6 and figures 7. All the comments made on the figures 6 at the end of section 2.3 also apply to the figures 7.



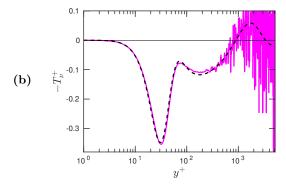


Fig. 5: The continuous line shows the opposite of the dimensionless turbulent diffusion term $-T_{\nu}^{+}$ (17) computed with the channel flow DNS of Lee & Moser (2015) at $Re_{\tau} = 543$ (a), 5186 (b). The dashed line shows the same term computed with our model (18).

2.5 Application to boundary layers

The boundary layer case differs from the channel and pipe flow cases in that the maximum value of y (resp. y^+) is not δ (resp. $\delta^+ = Re_{\tau}$) but, in principle, infinity. Moreover, the wake function W of boundary layers (10) differs significantly from the one of channel and pipe flows (9). The comparison of the figures 8 with the figures 6 and 7 shows similar behaviours in the ranges $y \in [0, \delta[$ i.e. $y^+ \in [0, \delta^+[$, whereas there are differences in the outer region. At $y = \delta$, i.e. the centerplane in channels or the pipe axis in pipes, the function T_{ν} should present a vanishing slope for symmetry reasons, as confirmed by the figures 6h and 7h; note that the outer term $-D_{\nu o}^+$ plays an important role there. In boundary layers, one does not expect a similar property, but that T_{ν} should approach 0 as $y \to \infty$. This is what suggests the figure 8h, and what would confirm a figure drawn with a larger interval of the abscissas (not shown): for all the Reynolds numbers implied, that range from 543 to 80000, $|T_{\nu}^+| < 10^{-3}$ as soon as $y > 1.32\delta$.

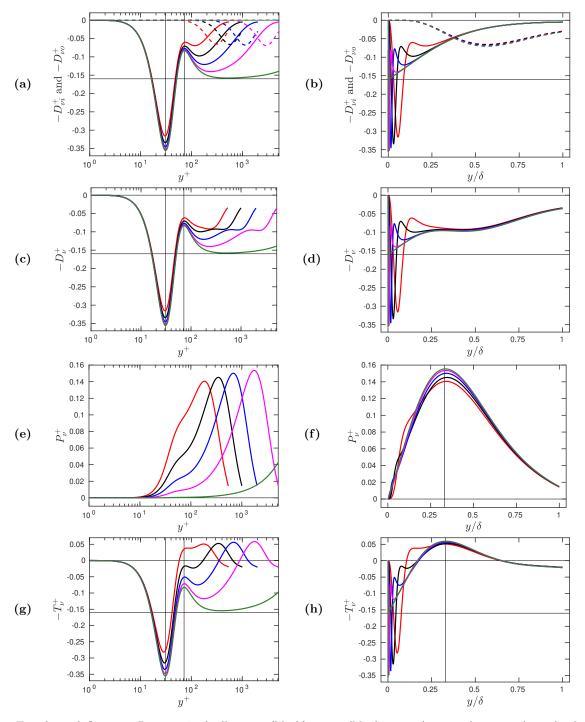


Fig. 6: For channel flows at $Re_{\tau} = 543$ (red), 1001 (black), 1995 (blue), 5186 (magenta), 80000 (green), the various contributions to $-T_{\nu}^{+}$ (18) and their sum. (a,b) $-D_{\nu i}^{+}$ with the continuous, $-D_{\nu o}^{+}$ with the dashed lines. (c,d) $-D_{\nu}^{+}$. (e,f) P_{ν}^{+} . (g,h) $-T_{\nu}^{+}$. On (a,c,g) the vertical lines are at $y^{+} = 31$ and 72; on (f,h) they are at $y = 0.33\delta$. On (a,b,c,d,g,h) the horizontal lines are at $-T_{\nu}^{+} = 0$ and $-\kappa^{2}$.

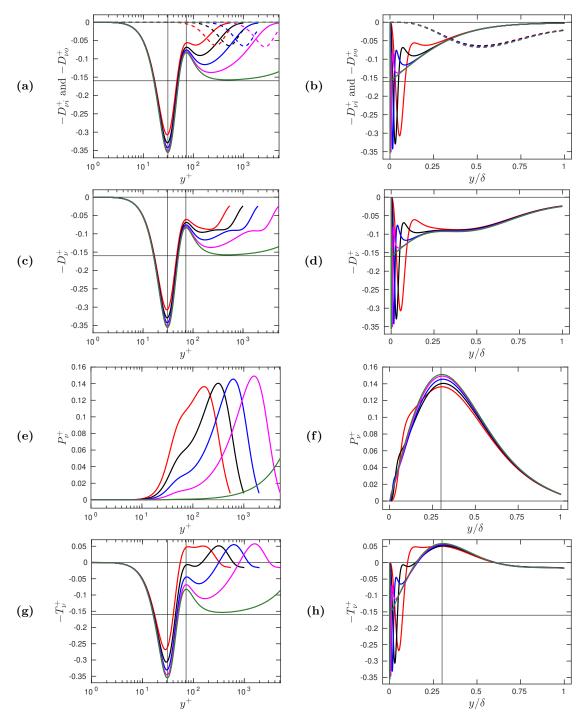


Fig. 7: Same as figure 6, but for pipe flows. On (f,h) the vertical lines are at $y = 0.3\delta$.

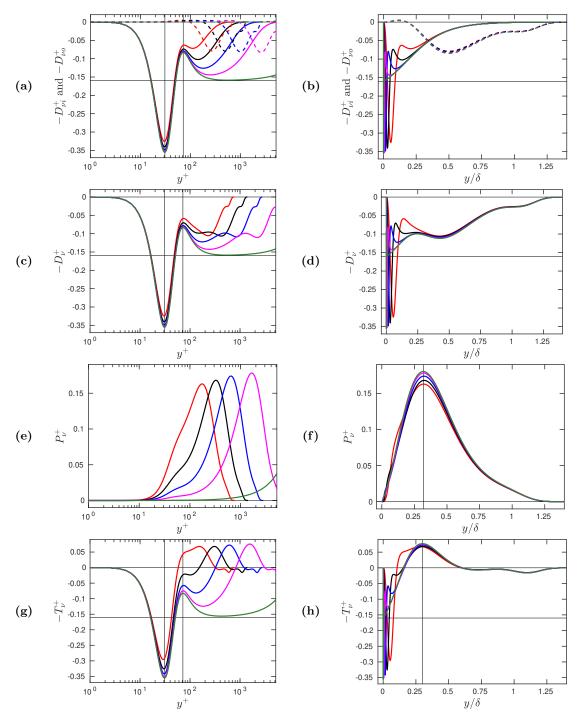


Fig. 8: Same as figure 6, but for boundary layers; in all graphs $1 \le y^+ \le 1.4\delta^+$. On (f) the vertical line is at $y = 0.32\delta$, on (h) it is at $y = 0.3\delta$.

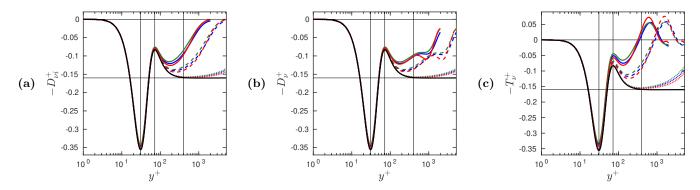


Fig. 9: The black curves show $-D_{\nu ia}^+$, see (19). The blue curves for channel flow, green curves for pipe flow, red curves for boundary layer show, in (a) $-D_{\nu i}^+$, (b) $-D_{\nu}^+$, (c) $-T_{\nu}^+$ for $Re_{\tau}=1995$ (continuous curves), 5186 (dashed curves) and 80000 (dotted curves). The vertical lines are at $y^+=31$, 72 and 400, the horizontal lines are at $-T_{\nu}^+=0$ and $-\kappa^2$.

2.6 Asymptotic structure of the near-wall dissipation

The figures 6acg, 7acg and 8acg show that, as $Re_{\tau} \to \infty$, the dissipation term $D_{\nu i}$ dominates the eddy-viscosity budget in the near-wall region, that it scales with y^+ , and approaches in inner units a universal asymptotic profile. This profile is obtained by replacing, in the expression (5) of the eddy-viscosity, which appears at the power 2 in $D_{\nu i}$ (13a), the wake function W by 1, since then the wake region goes to infinity in inner scaling: at fixed y^+ , $y/\delta = y^+/Re_{\tau} \to 0$ as $Re_{\tau} \to \infty$. This yields, as a relevant approximation of $D_{\nu i}$, the asymptotic dissipation function

$$D_{\nu ia} = \kappa^2 \frac{\nu^2}{L_{\nu K}^2} \frac{(1/S_{12}^+ - 1)^2}{f^2} \quad \text{or} \quad D_{\nu ia}^+ = \kappa^2 \frac{(1/S_{12}^+ - 1)^2}{L_{\nu K}^{+2}} \frac{1}{f^2} . \tag{19}$$

It is universal in that it does not depend on the flow case, but only on S_{12}^+ , see the equations (14) and (16). Moreover, $D_{\nu ia}^+$ considered as a function of y^+ also does not depend on Re_τ . As $y^+ \to \infty$, since $1/S_{12}^+ - 1$ and $L_{\nu K}^+$ approach κy^+ , whereas $f \to 1$, one has $D_{\nu ia}^+ \to \kappa^2$, in agreement with the expression of the log-layer eddy viscosity $\nu^+ = \kappa y^+$. This is visible on the figures 9; precisely, $|D_{\nu ia}^+ - \kappa^2| < 10^{-3}$ as soon as $y^+ \ge 400$. The colored curves in the figures 9 confirm that, at fixed y^+ , $D_{\nu i}$, D_{ν} and T_{ν} approach, as $Re_\tau \to \infty$, $D_{\nu ia}$, whatever the flow case. From a physical point of view, these results suggest that near-wall dissipation is due to universal near-wall motions.

2.7 Asymptotic structure of the production

The figures 6ef, 7ef and 8ef show that, as $Re_{\tau} \to \infty$, the production of the eddy viscosity vanishes in the near-wall region, scales with y/δ , and approaches asymptotic profiles that depend only on the flow case. These profiles are obtained by replacing, in the second expression of P_{ν} (13b), transformed in inner units,

$$P_{\nu}^{+} = \kappa \frac{\nu^{+}}{S_{12}^{+} L_{nK}^{+} \delta^{+}} (-4W') , \qquad (20)$$

the eddy viscosity ν^+ , the strain rate S_{12}^+ and the von Karman length-scale L_{vK}^+ by their approximations valid as $y^+ \to \infty$, i.e. $\kappa y^+ W$, $1/\kappa y^+$ and κy^+ respectively, see the discussions after equations (5-8) for ν^+ and S_{12}^+ , equations (14-15) for L_{vK}^+ . This yields the asymptotic profiles

$$P_{\nu a}^{+} = \kappa^{2} \frac{y}{\delta} (-4WW') \quad \text{or} \quad P_{\nu a} = \kappa^{2} u_{\tau}^{2} \frac{y}{\delta} (-4WW') .$$
 (21)

The first equation shows that $P_{\nu a}^+$ is, for a fixed flow case, a function of y/δ only, because the wake function W depends only on y/δ , see equations (9-10). The figures 10abc confirm that, at fixed y/δ , P_{ν}^+ approaches $P_{\nu a}^+$ as $Re_{\tau} \to \infty$. From a physical point of view, these results suggest that production is due to large-scale outer motions. The comparison between the vertical scales of the figures 10abc also suggest that these motions contribute more efficiently to the production of ν_t in the boundary layer than in the other flows. This is probably related to the fact that the boundary layer is in principle unbounded in the wall-normal direction, contrarily to channel and pipe flows.

2.8 Asymptotic structure of the dissipation in the outer region

The figures 6bd, 7bd and 8bd show that, as $Re_{\tau} \to \infty$, except in a narrow near-wall region, the dissipation terms $D_{\nu i}^+$, $D_{\nu o}^+$ and their sum D_{ν}^+ scale with y/δ , and approach asymptotic profiles that depend only on the flow case.

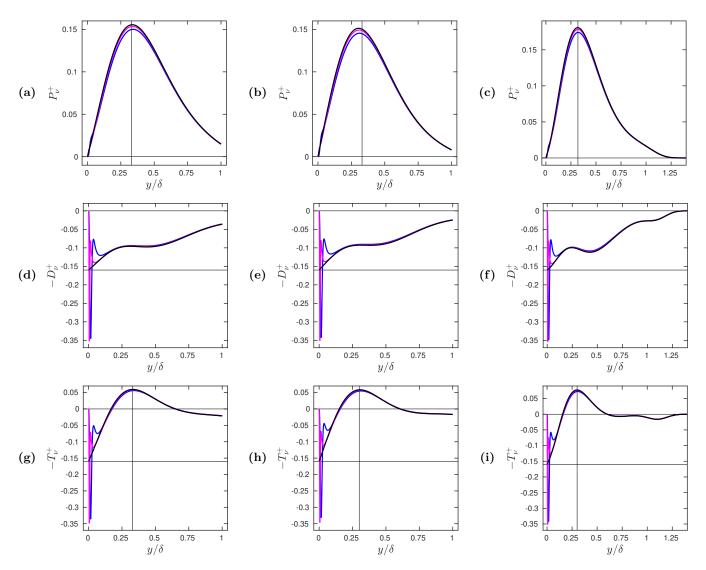


Fig. 10: For channel flow $(\mathbf{a},\mathbf{d},\mathbf{g})$, pipe flow $(\mathbf{b},\mathbf{e},\mathbf{h})$, boundary layer $(\mathbf{c},\mathbf{f},\mathbf{i})$. In $(\mathbf{a},\mathbf{b},\mathbf{c})$ the black curves show $P_{\nu a}^+$, see (21), the blue (resp. magenta) curves P_{ν}^+ for $Re_{\tau} = 1995$ (resp. 5186). The vertical lines are at $y/\delta = 0.33$ (a), 0.3 (b), 0.32 (c). In $(\mathbf{d},\mathbf{e},\mathbf{f})$, the black curves show $-D_{\nu a}^+$, see (23), the blue (resp. magenta) curves $-D_{\nu}^+$ for $Re_{\tau} = 1995$ (resp. 5186). In $(\mathbf{g},\mathbf{h},\mathbf{i})$, the black curves show $-T_{\nu a}^+$, see (24), the blue (resp. magenta) curves $-T_{\nu}^+$ for $Re_{\tau} = 1995$ (resp. 5186). In $(\mathbf{d}-\mathbf{i})$ the horizontal lines are at $-T_{\nu}^+ = 0$ and $-\kappa^2$. The vertical lines are at $y/\delta = 0.33$ (g), 0.3 (h), 0.3 (i).

These profiles are obtained by starting from the expressions (13a) and (13c), transformed in inner units,

$$D_{\nu}^{+} = \kappa^{2} \frac{\nu^{+2}}{L_{\nu K}^{+2}} \frac{1}{f^{2}} + \frac{1}{\delta^{+2}} (1/S_{12}^{+} - 1)^{2} (W'^{2} + WW'') , \qquad (22)$$

and applying the approximations that led from (20) to (21), plus $f \simeq 1$. This yields the asymptotic profiles

$$D_{\nu a}^{+} = \kappa^{2} W^{2} + \kappa^{2} \left(\frac{y}{\delta}\right)^{2} (W'^{2} + WW'') \quad \text{or} \quad D_{\nu a} = \kappa^{2} u_{\tau}^{2} W^{2} + \kappa^{2} u_{\tau}^{2} \left(\frac{y}{\delta}\right)^{2} (W'^{2} + WW'') , \qquad (23)$$

where the first (resp. second) term corresponds to the asymptotic profile of $D_{\nu i}^+$ or $D_{\nu i}$ (resp. $D_{\nu o}^+$ or $D_{\nu o}$). Similar to $P_{\nu a}^+$ (21), $D_{\nu a}^+$ is, for a fixed flow case, a function of y/δ only. The figures 10def confirm that, at fixed y/δ , D_{ν}^+ approaches $D_{\nu a}^+$ as $Re_{\tau} \to \infty$.

2.9 Asymptotic structure of the turbulent diffusion term in the outer region

Obviously

$$-T_{\nu a}^{+} = P_{\nu a}^{+} - D_{\nu a}^{+} \tag{24}$$

yields the asymptotic profiles of the opposite of the turbulent diffusion term in the outer region, as proven by the figures 10ghi.

2.10 Overview of the physical properties of the exact ν_t - equation

From the results of the sections 2.2 to 2.9, we can list the following physical properties of the exact ν_t - equation.

- (D0) The dissipation term $-D_{\nu i}$, that mainly scales with y^+ , dominates all the other terms in the near-wall region.
- (D1) As $Re_{\tau} \to \infty$, $D_{\nu i}^+$ converges to a universal function $D_{\nu ia}^+(y^+)$, given by the equation (19), displayed on the figures 9, that has a maximum max $D_{\nu ia}^+ \simeq 0.36$ around $y^+ = 31$.
- (D2) For larger values of y^+ , $D^+_{\nu ia}$ has a minimum min $D^+_{\nu ia} \simeq 0.08$ around $y^+ = 72$, and reaches a log-layer plateau $D^+_{\nu ia} = \kappa^2$ as soon as $y^+ \gtrsim 400$.

 All this suggests that dissipation of the eddy-viscosity is mainly due to universal near-wall motions.
- (T0) Beyond the log-layer plateau in terms of values of y, and as $Re_{\tau} \to \infty$, the opposite of the turbulent diffusion term $-T_{\nu}^{+}$ converges to asymptotic profiles $-T_{\nu a}^{+}$, given by the equation (24), that depend on y/δ and on the flow case only. These asymptotic profiles, displayed on the figures 10ghi, start at in the near-wall region the log-layer value $-T_{\nu a}^{+} = -\kappa^{2}$.
- (P) For larger values of y, the functions $-T_{\nu a}^+$ show a maximum due to the production term around $y = 0.3\delta$. The scaling with y/δ and the position of this maximum suggest that production of the eddy-viscosity is due to large-scale outer motions.
- (T1) As $y \to \delta$ (resp. $y \to \infty$), the functions $-T_{\nu a}^+$ converge to a small negative value (resp. vanishing value) in the channel and pipe flow cases (resp. boundary layer case).

The properties (P) contrast strongly with the ones of the production of the turbulent kinetic energy k, which scales with y^+ and peaks around $y^+ = 11$, as shown for instance in the supplementary material to Heinz (2019). These robust properties will now be used to study existing models of the eddy-viscosity equation.

3 Evaluation of other eddy-viscosity models

From now on, the focus is on channel flows, which are very well documented, and where the geometry is the most simple. Our aim is a review of existing models of the eddy-viscosity equation, by a test of the properties listed in the section 2.10 through relevant plots.

3.1 About the model of Spalart & Allmaras (1994)

The high-Reynolds number eddy-viscosity equation (4) of Spalart & Allmaras (1994) reads, for channel flow,

$$\sigma_S \frac{\partial \nu_t}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \nu_t}{\partial y} \right) + c_{b2} \left(\frac{\partial \nu_t}{\partial y} \right)^2 + \sigma_S c_{b1} S \nu_t - \sigma_S c_{w1} f_w \left(\frac{\nu_t}{y} \right)^2, \tag{25}$$

with the same notations, except for σ_S which stands for the σ of Spalart & Allmaras (1994), and

$$f_w = g \left(\frac{1 + c_{w3}^6}{g^6 + c_{w3}^6}\right)^{1/6}, \quad g = r + c_{w2}(r^6 - r), \quad r = \frac{\nu_t}{S \kappa_S^2 y^2},$$
 (26)

 $\sigma_S = 2/3$, $c_{b1} = 0.1355$, $c_{b2} = 0.622$, $\kappa_S = 0.41$, $c_{w1} = c_{b1}/\kappa_S + (1 + c_{b2})/\sigma_S$, $c_{w2} = 0.3$, $c_{w3} = 2$; their von Karman constant κ_S differs slightly from ours (8). With the definitions (12) and (17) of the turbulent diffusion term in physical and dimensionless forms, we identify their model for $-T_{\nu}^+$,

$$-T_{\nu S}^{+} = \sigma_{S} c_{b1} S^{+} \nu^{+} + c_{b2} \left(\frac{\partial \nu^{+}}{\partial y^{+}}\right)^{2} - \sigma_{S} c_{w1} f_{w} \left(\frac{\nu^{+}}{y^{+}}\right)^{2}$$
 (27)

with, in particular,

$$r = \frac{\nu^+}{S^+ \kappa_S^2 y^{+2}} \,. \tag{28}$$

To compute S^+ accurately, for the evaluation of the first and third terms in (27), the equation (7) of Heinz (2018), that takes into account wake effects, is used. The model (5) is used on the other hand to compute ν^+ . As a first test of Spalart & Allmaras (1994) model, plots in the near-wall region, with y^+ as the abscissa, are displayed on the figures 11ab. According to the properties (D0,D1) of the exact ν_t – equation, the colored curves of the figure 11a should approach the black curve showing $-D^+_{\nu ia}$. This is not at all the case, and more seriously the model of Spalart

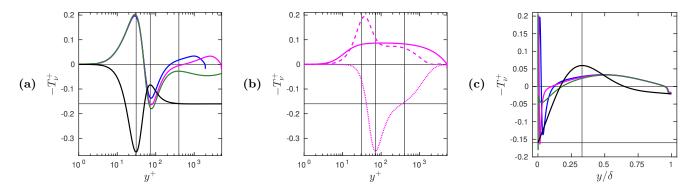


Fig. 11: Evaluation of the model of Spalart & Allmaras (1994) in channel flow. In (a), the black curve shows $-D_{\nu ia}^+$, see (19), the colored curves show $-T_{\nu S}^+$, see (27), for $Re_{\tau}=1995$ (blue), 5186 (magenta) and 80000 (green). In (b), for $Re_{\tau}=5186$, the curves show $\sigma_S c_{b1} S^+ \nu^+$ (continuous), $c_{b2} (\partial \nu^+ / \partial y^+)^2$ (dashed), $-\sigma_S c_{w1} f_w (\nu^+ / y^+)^2$ (dotted). In (a,b) the vertical lines are at $y^+=31$, 72 and 400. In (c), the black curve shows $-T_{\nu a}^+$, see (24), the colored curves show $-T_{\nu S}^+$ for $Re_{\tau}=1995$ (blue), 5186 (magenta) and 80000 (green), the vertical line is at $y=0.33\delta$. In (a,b,c) the horizontal lines are at $-T_{\nu}^+=0$ and $-\kappa^2$.

& Allmaras (1994) predicts a near-wall production peak where there should be a near-wall dissipation peak. The figure 11b confirms that the dissipation peak of the model occurs at too large values of y^+ , and also shows that the inclusion of the differential production term proportional to $(\partial \nu^+/\partial y^+)^2$ does not help. Thus the high-Reynolds number model of Spalart & Allmaras (1994) does not describe correctly the physics of the ν_t – equation in the near-wall region. It is also noticeable that no log-layer plateau appears in Spalart & Allmaras' model, even at $Re_{\tau} = 80000$, contrarily to what shows the exact ν_t – equation: compare the figures 9c and 11a. This raises questions, since the classical 'log-layer equilibrium' has been used in the derivation of the model of Spalart & Allmaras (1994) to relate c_{w1} to the other coefficients.

Plots with y/δ as the abscissa are displayed on the figure 11c. The outer production peak of $-T_{\nu S}^+$, which was already visible for the lowest values of Re_{τ} in the figure 11a, seems, on the figure 11c, to scale with y/δ . Moreover, the value of this outer maximum has the correct magnitude. It is also remarkable that the values of $-T_{\nu S}^+$ at $y=\delta$ are quite correct. Thus, in the outer region there is a qualitative and even semi-quantitative agreement between the model of Spalart & Allmaras (1994) and the properties (T,P) of the exact ν_t — equation. However, the fact that the colored curves of the figure 11c differ significantly from the black curve, especially for the largest value of Re_{τ} , reveals that there are, in the outer region, quantitatively significant differences between the model and the exact theory.

In summary, the model of Spalart & Allmaras (1994) appears to be of poor quality in the near-wall region, and of rather good relevance in the outer region.

3.2 About the model of Yoshizawa et al. (2012)

The model of Yoshizawa et al. (2012) implies three turbulent fields: the eddy viscosity ν_t , the turbulent kinetic energy k and the turbulent dissipation ϵ . It also proposes an eddy-viscosity equation, which was shown to yield a better model of ν_t than the eddy-viscosity formula of the standard $k-\epsilon$ model in some specific cases. The high-Reynolds number form of the ν_t – equation (60) of Yoshizawa et al. (2012) reads, for channel flow,

$$\sigma_Y \frac{\partial \nu_t}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \nu_t}{\partial y} \right) + \sigma_Y C_{\nu P} f_{\nu} k - \sigma_Y C_{\nu \epsilon} \frac{\nu_t}{\tau} , \qquad (29)$$

with the same notations, except for σ_Y which stands for the σ_{ν} of Yoshizawa et al. (2012), and

$$f_{\nu} = \left(1 - \exp\left(-\frac{y^*}{14}\right)\right)^2 \left(1 + \frac{5}{R_t^{3/4}} \exp\left(-\left(\frac{R_t}{200}\right)^2\right)\right), \quad \tau = \frac{k}{\epsilon \Lambda}, \quad \Lambda = \sqrt{1 + 2(C_S + C_\Omega)\left(\frac{kS}{\epsilon}\right)^2}, \quad (30)$$

 $\sigma_Y = 3$, $C_{\nu P} = 4/15$, $C_{\nu \epsilon} = 3.5$, $C_S = 0.015$, $C_{\Omega} = 0.02C_S$, $y^* = (\nu \epsilon)^{1/4} y/\nu$, $R_t = k^2/(\nu \epsilon)$. Their model for $-T_{\nu}^+$ reads therefore

$$-T_{\nu Y}^{+} = \sigma_{Y} C_{\nu P} f_{\nu} k^{+} - \sigma_{Y} C_{\nu \epsilon} \frac{\epsilon^{+}}{k^{+}} \Lambda \nu^{+}$$
(31)

with, in particular,

$$k^{+} = \frac{k}{u_{-}^{2}}, \quad \epsilon^{+} = \frac{\nu \epsilon}{u_{-}^{4}}, \quad y^{*} = (\epsilon^{+})^{1/4} y^{+}, \quad R_{t} = \frac{k^{+2}}{\epsilon^{+}}, \quad \frac{kS}{\epsilon} = \frac{k^{+}S^{+}}{\epsilon^{+}}.$$
 (32)

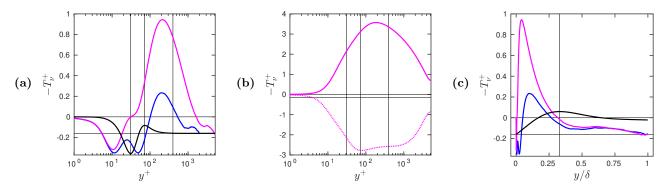


Fig. 12: Evaluation of the model of Yoshizawa et al. (2012) in channel flow, using the DNS of Lee & Moser (2015). In (a), the black curve shows $-D_{\nu ia}^+$, see (19), the colored curves show $-T_{\nu Y}^+$, see (31), for $Re_{\tau}=1995$ (blue) and 5186 (magenta). In (b), for $Re_{\tau}=5186$, the curves show $\sigma_Y C_{\nu P} f_{\nu} k^+$ (continuous), $-\sigma_Y C_{\nu \epsilon} \frac{\epsilon^+}{k^+} \Lambda \nu^+$ (dotted). In (a,b) the vertical lines are at $y^+=31$, 72 and 400. In (c), the black curve shows $-T_{\nu a}^+$, see (24), the colored curves show $-T_{\nu Y}^+$ for $Re_{\tau}=1995$ (blue) and 5186 (magenta), the vertical line is at $y=0.33\delta$. In (a,b,c) the horizontal lines are at $-T_{\nu}^+=0$ and $-\kappa^2$.

Since the fields k and ϵ are needed in this model, we use the DNS data of Lee & Moser (2015) to test it. The reduced eddy viscosity ν^+ is also extracted from the DNS. Plots with y^+ as the abscissa are displayed on the figures 12ab. A good property of the model of Yoshizawa et al. (2012) is that it presents a dissipation peak in the near-wall region, that has the correct magnitude, and seems to scale with y^+ . Thus the properties (D0,D1) of the theory are qualitatively fulfilled. However, quantitatively, the dissipation peak of $-T_{\nu Y}^+$ comes in too early in terms of y^+ values: for $Re_{\tau} = 5186$, $-T_{\nu Y}^+$ shows a minimum around $y^+ = 10$ instead of $y^+ = 31$ for the minimum of $-D_{\nu ia}^+$. For larger values of y^+ , the model of Yoshizawa et al. (2012) appears to be too productive, and there is no log-layer plateau. Plots with y/δ as the abscissa are displayed on the figure 12c. These plots confirm that the model of Yoshizawa et al. (2012) is too productive, moreover the scaling with y/δ does not show up, i.e., the properties (T,P) of the theory are not fulfilled.

In summary, the model of Yoshizawa et al. (2012) appears to be of rather good relevance in the near-wall region, but of poor quality in the outer region. Moreover, this model shows a too-strong Reynolds-number dependence.

3.3 About the SAS model of Menter et al.

The high-Reynolds number $k - \sqrt{k}\ell$ SAS model of Menter et al. has been introduced in Menter et al. (2006) and discussed in more details in Menter & Egorov (2010). The two turbulent fields are k and $\sqrt{k}\ell$ with ℓ the turbulent length scale. The product $\sqrt{k}\ell$ is up to a constant factor the eddy viscosity ν_t , hence this model may be presented as a $k - \nu_t$ model. After multiplication by $\rho^{-1} c_\mu^{1/4} \sigma_M$, with the notations of Menter et al. (2006), except for σ_M which stands for the σ_Φ of Menter et al. (2006), their equation (7) reads, for channel flow,

$$\sigma_M \frac{\partial \nu_t}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \nu_t}{\partial y} \right) + \sigma_M \frac{\nu_t^2 S^2}{k} \left(\zeta_1 - \zeta_2 \left(\frac{\ell}{\ell_{vK}} \right)^2 \right) - \sigma_M c_\mu^{1/4} \zeta_3 k - 6\sigma_M f_\Phi \frac{\nu \nu_t}{y^2} , \qquad (33)$$

with

$$\ell = c_{\mu}^{-1/4} \frac{\nu_t}{\sqrt{k}}, \quad \ell_{vK} = \kappa_M \left| \frac{S}{\partial S/\partial y} \right|, \quad f_{\Phi} = \frac{1 + c_{d1}\xi}{1 + \xi^4}, \quad \xi = \frac{\sqrt{0.3 \ k} \ y}{20\nu}, \tag{34}$$

 $\sigma_M = 2/3$, $c_\mu = 0.09$, $\kappa_M = 0.41$, $\zeta_1 = 0.8$, $\zeta_2 = 1.47$, $\zeta_3 = 0.0288$, $c_{d1} = 4.7$. The definition of the von Karman length scale in (34) agrees perfectly with our definition (15), except for the different value of the von Karman constant; for the sake of brevity we use the same notation ℓ_{vK} , whereas in this section κ_M is used instead of κ for the computation of the SAS term, proportional to $(\ell/\ell_{vK})^2$, in (33). Importantly, we include as the last term of (33) the 'viscous sublayer model' term of Menter *et al.* (2006), defined in their equation (10), since we want a ν_t - equation as accurate as possible in the near-wall region. The dimensional factor in this term, $\nu \nu_t/y^2$, has similarities with the one of the last term of the ν_t - equation (25) of Spalart & Allmaras (1994), ν_t^2/y^2 , with, however, one eddy viscosity replaced by the fluid viscosity. Menter's SAS model for $-T_{\nu}^+$ reads therefore

$$-T_{\nu M}^{+} = P_{\nu M}^{+} \left(\zeta_{1} - \zeta_{2} \left(\frac{\ell^{+}}{\ell_{\nu K}^{+}} \right)^{2} \right) - \sigma_{M} c_{\mu}^{1/4} \zeta_{3} k^{+} - 6\sigma_{M} f_{\Phi} \frac{\nu^{+}}{y^{+2}}$$
 (35)

with

$$P_{\nu M}^{+} = \sigma_{M} \frac{\nu^{+2} S^{+2}}{k^{+}}, \quad \ell^{+} = c_{\mu}^{-1/4} \frac{\nu^{+}}{\sqrt{k^{+}}}, \quad \xi = \frac{\sqrt{0.3 \ k^{+}} \ y^{+}}{20}.$$
 (36)

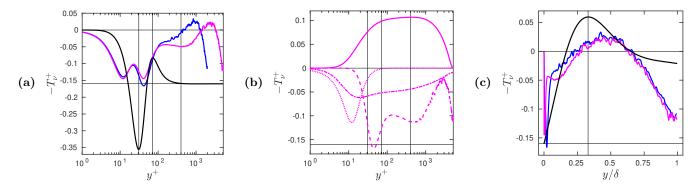


Fig. 13: Evaluation of the SAS model of Menter et al. (2006) in channel flow, using the DNS of Lee & Moser (2015). In (a), the black curve shows $-D_{\nu ia}^+$, see (19), the colored curves show $-T_{\nu M}^+$, see (35), for $Re_{\tau}=1995$ (blue) and 5186 (magenta). In (b), for $Re_{\tau}=5186$, the curves show ζ_1 $P_{\nu M}^+$ (continuous), $-\zeta_2$ $P_{\nu M}^+$ ($\ell/\ell_{\nu K}$)² (dashed), $-\sigma_M c_{\mu}^{1/4} \zeta_3$ k^+ (dashedt), $-6\sigma_M$ f_{Φ} ν^+/y^{+2} (dotted). In (a,b) the vertical lines are at $y^+=31$, 72 and 400. In (c), the black curve shows $-T_{\nu a}^+$, see (24), the colored curves show $-T_{\nu M}^+$ for $Re_{\tau}=1995$ (blue) and 5186 (magenta), the vertical line is at $y=0.33\delta$. In (a,b,c) the horizontal lines are at $-T_{\nu}^+=0$ and $-\kappa^2$.

Importantly, we focus first on the SAS model without limiters, since channel flows do not present adverse pressure gradients or stagnation regions. The DNS data of Lee & Moser (2015) are used to test this model. Finite differences are used to compute $\partial S^+/\partial y^+$ to estimate ℓ_{vK}^+ . Plots with y^+ as the abscissa are displayed on the figures 13ab. The figure 13a shows that there is a near-wall dissipation peak in the SAS model, that scales in y^+ , i.e. the properties (D0,D1) of the theory are qualtitatively fulfilled. However, the minimum value of $-T_{vM}^+$ corresponding to this dissipation peak is too small, and this peak comes in too early in terms of y^+ values: for $Re_{\tau}=5186$, $-T_{vY}^+$ shows a minimum around $y^+=12$ instead of $y^+=31$ for the minimum of $-D_{via}^+$. For slightly larger values of y^+ , the SAS model displays for the two cases shown the good behaviour with a maximum and then a decrease. However, no log-layer plateau shows up in the SAS profiles, at least for $Re_{\tau} \lesssim 5200$, whereas the classical 'log-layer equilibrium' has been used in the derivation of the model of Menter *et al.* (2006) to relate ζ_2 to the other coefficients. The figure 13b showing the different contributions to $-T_{vM}^+$ proves that the SAS term in (35) plays a quite important role, both around $y^+=41$ where $-T_{vM}^+$ has its second minimum, and near the centerplane, at $y^+=\delta^+$, where it imposes a rather large negative value.

Plots with y/δ as the abscissa are displayed on the figure 13c. They show that $-T_{\nu M}^+$ scales with y/δ in the outer region, with a production peak around $y=0.42\delta$, and then a decrease towards a negative value at the centerplane. Thus the properties (T,P) of the theory are qualitatively fulfilled.

In summary, of the three models tested up to now, the SAS model without limiters shows the best qualitative agreement with the theory. It also agrees semi-quantitatively with the theory, as shows the comparison between the figures 11c, 12c and 13c: only the latest shows model curves (the coloured curves) that live in the interval of values of the ordinate swept by the exact asymptotic profile (the black curve).

In order to better analyze the SAS models, the figures 14abc display the two length scales implied, and their ratio. The figures 14ab suggest that both length scales, scaled by δ , scale with y/δ , except in a narrow near-wall region for ℓ_{vK} . Since ℓ_{vK}/δ is computed from the DNS as

$$\frac{\ell_{vK}^+}{\delta^+} = \frac{\kappa_M}{\delta^+} \left| \frac{S^+}{\partial S/\partial y^+} \right|, \tag{37}$$

because both S^+ and $\partial S^/ \partial y^+$ become quite small in the outer region for large Re_{τ} , the DNS noise is amplified there. This explains the oscillations in the figures 14bc, that also blur the profiles of the figures 13. Smoother profiles of ℓ_{vK} may be obtained from the exact model of Heinz (2018, 2019), and have been shown on the figure 3. The laws that result from the classical log-layer theory,

$$\ell = \ell_{vK} = \kappa_M y \,, \tag{38}$$

are relevant in a narrow near-wall region for ℓ and in a larger region for ℓ_{vK} , which otherwise vanishes at the centerplane (as already discussed after equation 15). The ratio ℓ/ℓ_{vK} displayed on the figure 14c shows consequently a near-wall peak of maximum value of order 1, which locates somehow the log-layer region. It then decays, since ℓ_{vK} increases first faster than ℓ , and finally increases again and diverges as $y \to \delta$. Obviously, the large values of ℓ/ℓ_{vK} near the centerplane play a role in the too large value of $-T_{vM}^+$ in this region, see the figure 13c.

From these observations, it seems relevant to test also the SAS model with length-scale limiters, since these limiters have been defined 'in order of avoiding overly large or small values of the length scale ratio' ℓ/ℓ_{vK} , as explained by

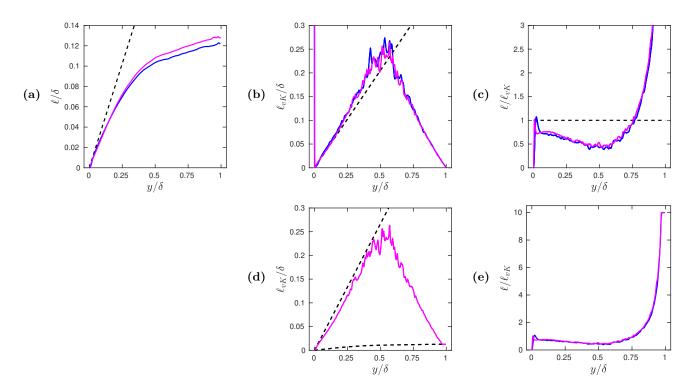


Fig. 14: For channel flow, using the DNS of Lee & Moser (2015) at $Re_{\tau} = 1995$ (blue) and 5186 (magenta), the turbulent length scale ℓ (a), the von Karman length scale ℓ_{vK} (b), their ratio (c). In (a,b) the dashed line shows the log-layer length scale $\kappa_M y^+$. The effects of the length scale limiters (39) is shown on (d) for $Re_{\tau} = 5186$, with the limiters shown by the dashed curves, and on (e) for both Reynolds numbers.

Menter et al. (2006) at the level of their equation (12). These length-scale limiters are defined by

$$\ell/c_{\ell 1} < \ell_{vK} < c_{\ell 2} \kappa_M y \tag{39}$$

with $c_{\ell 1}=10$, $c_{\ell 2}=1.3$. The minimum and maximum limiters are displayed on the figure 14d for the highest Reynolds number available in the DNS database of Lee & Moser (2015). Obviously the maximum limiter stays inactive, whereas the minimum limiter is active only in a narrow region near the centerplane. Consequently the length scale ratio is only limited in the same region, as displayed on the figure 14e. This will saturate the minimum of $-T_{\nu M}^+$ at the centerplane only marginaly, with reference to the figure 13c. From this point of view, a lower value of $c_{\ell 1}$ would help.

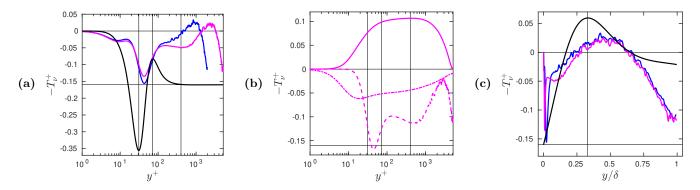


Fig. 15: Same as figure 13, but without the 'viscous sublayer model' term.

3.4 Q7 Test of the SAS model without the VSM relevant?

• Q7 (Emmanuel, July 17): It is easy to remove the 'viscous sublayer model' (VSM) term from $-T_{\nu M}^+$ (35) and to redraw the figures 13 without this term. This yields the figures 15. This model is naturally equivalent to the one with VSM in the outer region. In the inner region, it is somehow poorer since the first dissipation peak seems to be too much Reynolds-dependent... but one could imagine other comments!..

Should we also show this test and figure?

• R7 (Stefan, August 21): yes, 'to be on the safe side'!

3.5 Q8 Test of the SAS model with the ν_t limiters relevant?

• Q8 (Emmanuel, July 17): In the equations (11) of Menter et al. (2006) they define rather complex ν_t limiters.

Should we also test the SAS model with these ν_t limiters?

• R8 (Stefan, August 21): yes, 'to be on the safe side'!

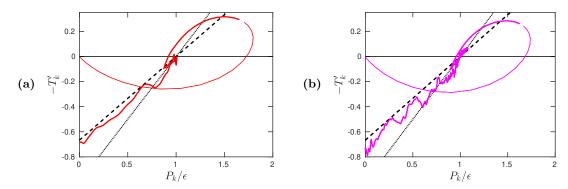


Fig. A.1: The continuous line shows the opposite of the normalized turbulent diffusion term $-T'_k$ (A.2) vs the production-to-dissipation ratio P_k/ϵ for the channel flow DNS of Lee & Moser (2015) at $Re_{\tau} = 543$ (a), 5186 (b). The thin (resp. thick) part corresponds to the inner region $y^+ < 15$ (resp. outer region $y^+ > 15$). The dashed (resp. dotted) line shows the model (A.3) with $\sigma_k = 2/3$ (resp. 1).

Appendices

A Validation of the standard k - equation with channel flow DNS

In the standard, high-Reynolds number $k - \epsilon$ (Launder & Spalding 1974) and $k - \omega$ (Wilcox 1988) models, for channel flows, the closed k - equation reads

$$\sigma_k \frac{\partial k}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial k}{\partial y} \right) + \sigma_k \left(P_k - \epsilon \right) \tag{A.1}$$

with σ_k a model coefficient, and the production term

$$P_k = \nu_t S^2$$

according to the eddy-viscosity hypothesis, with the notations of section 2.1. The equation (A.1) is also used in the SAS model of Menter et al. (2006); Menter & Egorov (2010). Various values of σ_k are recommended: $\sigma_k = 1$ in Launder & Spalding (1974), 2 in Wilcox (1988), 2/3 in Menter et al. (2006). To discriminate between those, and confirm the relevance of the k - equation, we analyze the equation (A.1) with the approach of Heinz (2006). After division by ϵ , equation (A.1) states that the opposite of the dimensionless normalized turbulent diffusion term

$$-T'_{k} = -\frac{1}{\epsilon} \frac{\partial}{\partial y} \left(\nu_{t} \frac{\partial k}{\partial y} \right) \tag{A.2}$$

should be a linear function of the production-to-dissipation ratio P_k/ϵ ,

$$-T_k' = \sigma_k \left(P_k / \epsilon - 1 \right). \tag{A.3}$$

This prediction is tested on the channel flow DNS data of Lee & Moser (2015) on the figures A.1. Note that Lee & Moser (2015) offer in their figure 7 plots of P_k/ϵ vs y^+ . The eddy viscosity ν_t is computed according to its definition (3), and the derivatives by y are computed by finite differences. The figures A.1 show that, in the outer region $y^+ > 15$, the DNS curves remain close to the line (A.3): this confirms the relevance of the high-Reynolds number k - equation (A.1), and supports Menter *et al.* (2006) in their choice $\sigma_k = 2/3$. The value of Launder & Spalding (1974), $\sigma_k = 1$, seems a bit too large, whereas the value of Wilcox (1988), $\sigma_k = 2$, seems clearly too large.

B Study of the standard ϵ and ω - equations with channel flow DNS

By analogy with (A.1), Launder & Spalding (1974) postulated in the $k-\epsilon$ model the ϵ - equation, for channel flows,

$$\sigma_{\epsilon} \frac{\partial \epsilon}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_{t} \frac{\partial \epsilon}{\partial y} \right) + \sigma_{\epsilon} \frac{\epsilon}{k} \left(C_{1} P_{k} - C_{2} \epsilon \right)$$
(B.1)

with $\sigma_{\epsilon} = 1.3$, $C_1 = 1.44$, $C_2 = 1.92$. After division by ϵ^2/k , equation (B.1) states that the opposite of the normalized turbulent diffusion term

$$-T'_{\epsilon} = -\frac{k}{\epsilon^2} \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \epsilon}{\partial y} \right) \tag{B.2}$$

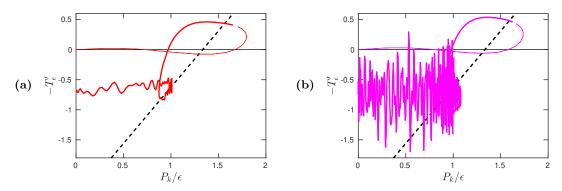


Fig. B.1: The continuous line shows the opposite of the normalized turbulent diffusion term $-T'_{\epsilon}$ (B.2) for the channel flow DNS of Lee & Moser (2015) at $Re_{\tau} = 543$ (a), 5186 (b). The thin (resp. thick) part corresponds to the inner region $y^+ < 15$ (resp. outer region $y^+ > 15$). The dashed line shows the standard model (B.3).

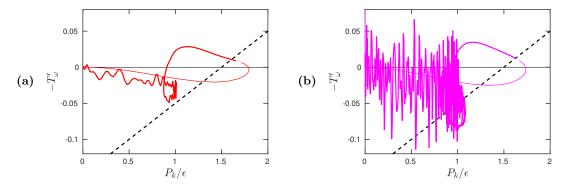


Fig. B.2: The continuous line shows the opposite of the normalized turbulent diffusion term $-T'_{\omega}$ (B.5) for the channel flow DNS of Lee & Moser (2015) at $Re_{\tau} = 543$ (a), 5186 (b). The thin (resp. thick) part corresponds to the inner region $y^+ < 15$ (resp. outer region $y^+ > 15$). The dashed line shows the standard model (B.6).

should be a linear function of P_k/ϵ ,

$$-T'_{\epsilon} = \sigma_{\epsilon} \left(C_1 P_k / \epsilon - C_2 \right). \tag{B.3}$$

This prediction is tested on the channel flow DNS of Lee & Moser (2015) on the figures B.1. Since ϵ becomes quite small near the centerplane (see e.g. the figure 9 of Heinz 2019), there the DNS noise is amplified: this region corresponds to the lower intersection of the curves of the figures B.1 with the axis $P_k/\epsilon = 0$ i.e. $P_k = 0$ (see the figure 7 of Lee & Moser 2015). The figures B.1, to be compared with the figures A.1, show DNS curves that do not align with the linear model (B.3), even if one considers only their outer-region part. The structure of the curves is highly nonlinear, therefore a change of the model constants σ_{ϵ} , C_1 and C_2 cannot solve this problem.

A similar flaw exists with the ω - equation of the standard $k-\omega$ model of Wilcox (1988). With a slight change of notation, to introduce a coefficient σ_{ω} that plays a role similar to the coefficients σ_k in (A.1) and σ_{ϵ} in (B.1), Wilcox' equation for

$$\omega = \epsilon/(\beta^* k)$$

reads, for channel flows,

$$\sigma_{\omega} \frac{\partial \omega}{\partial t} = 0 = \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \omega}{\partial y} \right) + \sigma_{\omega} \frac{\omega}{k} \left(\gamma P_k - \frac{\beta}{\beta^*} \epsilon \right)$$
 (B.4)

where $\sigma_{\omega} = 2$, $\gamma = 5/9$, $\beta = 3/40$, $\beta^* = 9/100$. After division by ω^2 , equation (B.4) states that

$$-T'_{\omega} = -\frac{1}{\omega^2} \frac{\partial}{\partial y} \left(\nu_t \frac{\partial \omega}{\partial y} \right) \tag{B.5}$$

should be a linear function of P_k/ϵ ,

$$-T'_{\omega} = \sigma_{\omega} \left(\beta^* \gamma P_k / \epsilon - \beta \right). \tag{B.6}$$

The figures B.2 show again a highly nonlinear structure of the channel flow DNS curves, that cannot fit a linear model such as (B.6), even in the outer region $y^+ > 15$.

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- V0.11 of June 18, 2020: memory of the section 4.3 About the model of Hamba (2013) with a study of its production term.
- V0.085 of June 4, 2020: memory of the section 3.8 QR6 on the BL case inspired from Spalart & Allmaras (1994) possible advection effects that might change the lhs of the ν_t eq.
- V0.055 of May 19, 2020: memory of the section 4.1 Comparison with Hamba (2013), of figure 9 for P_{ν} and $P_{\nu H}$, of QR5 regarding this comparison.
- V0.025 of May 7, 2020: memory of figure 5 for W'/W and $(W'^2 + WW'')/W^2$, QR3 regarding comparisons with DNS of pipe flow, QR4 regarding comparisons with DNS of BL, figure 10 for $-T_{\nu}^+$ et al. in BL with an extended range of y^+ .
- V0.015 of April 29, 2020: memory of QR1 regarding near-wall effects / the kinematic viscosity & QR2 regarding pipe flow / curvature effects.